

$$D_{12} = \text{Tr} \left\{ \begin{bmatrix} m_2 l_{s2} (l_{s1} + l_{s12}) & & & & \\ & m_2 l_{c2} (l_{c1} + l_{c2}) & & & \\ & & 0 & & \\ & & & 0 & \\ & & & & 0 \end{bmatrix} \right\}$$

$$\Rightarrow D_{12} = m_2 l^2 [\cos(\theta_1 - \theta_2) + c_1]$$

02.12.2002 / Monday

ANGULAR VELOCITY AND ACCELERATION:

"Robot Dynamics and Control", M.W. Spong, M. Vidyasagar, Wiley pp 50-56

Defining skew symmetric matrix:

$$S + S^T = 0$$

$$s_{ij} + s_{ji} = 0 \quad i, j = 1, 2, 3, \dots$$

$$s_{ii} = 0$$

$$S(\underline{s}) = \begin{bmatrix} 0 & -s_3 & s_2 \\ s_3 & 0 & -s_1 \\ -s_2 & s_1 & 0 \end{bmatrix} \quad \text{for } \underline{s} = s_1 i + s_2 j + s_3 k$$

$$S(i) = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -1 \\ 0 & 1 & 0 \end{bmatrix} \quad \text{for } i = \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix}$$

$$S(j) = \begin{bmatrix} 0 & 0 & 1 \\ 0 & 0 & 0 \\ -1 & 0 & 0 \end{bmatrix} \quad \text{for } j = \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix}$$

$$S(k) = \begin{bmatrix} 0 & -1 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \quad \text{for } k = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

Properties of Skew Symmetric Matrices:

i. The linearity property

For any two vectors  $\underline{a}$  and  $\underline{b} \in \mathbb{R}^3$  and scalars  $\alpha$  and  $\beta$

$$S(\alpha \underline{a} + \beta \underline{b}) = \alpha S(\underline{a}) + \beta S(\underline{b})$$

ii. For  $\underline{p} = \begin{bmatrix} p_x \\ p_y \\ p_z \end{bmatrix}$  and  $\underline{a} = \begin{bmatrix} a_x \\ a_y \\ a_z \end{bmatrix}$

$$S(\underline{a})\underline{p} = \underline{a} \times \underline{p}$$

iii. If  $\underline{a}, \underline{b} \in \mathbb{R}^3$  then

$$R(\underline{a} \times \underline{b}) = R\underline{a} \times R\underline{b}$$

If  $R$  is an orthogonal rotation matrix.

iv. For two vectors  $\underline{a}$  and  $\underline{b}$

$$\begin{aligned} R S(\underline{a}) R^T \underline{b} &= R(\underline{a} \times R^T \underline{b}) \quad \dots (i) \\ &= (R\underline{a}) \times \underbrace{(R R^T \underline{b})}_{\underline{I}} \quad \dots (ii) \\ &= (R\underline{a}) \times \underline{b} \\ &= S(R\underline{a}) \underline{b} \end{aligned}$$

Therefore,  $R S(\underline{a}) R^T = S(R\underline{a}) \rightarrow$  similarity transformation of  $S(\underline{a})$

For example,  $R = \text{Rot}(\theta)$

$$\text{Rot}(\theta) \cdot \text{Rot}^T(\theta) = \underline{I}$$

$$\underbrace{\frac{dR}{d\theta}}_{S^T} \cdot R^T(\theta) + R(\theta) \cdot \underbrace{\frac{dR^T(\theta)}{d\theta}}_{S^T} = 0$$

$$S \triangleq \frac{dR}{d\theta} \cdot R^T(\theta)$$

$$S^T \triangleq \left( \frac{dR}{d\theta} R^T(\theta) \right)^T = R(\theta) \cdot \frac{dR^T(\theta)}{d\theta}$$

$$\frac{dR}{d\theta} \cdot \underbrace{R^T(\theta) \cdot R(\theta)}_{\underline{I}} = S R(\theta)$$

$$\frac{dR}{d\theta} = S R(\theta)$$

Example: Let  $R = R(\theta) = \text{Rot}(x, \theta)$

$$S = \frac{dR}{d\theta} R^T(\theta) = \begin{bmatrix} 0 & 0 & 0 \\ 0 & -\sin\theta & \cos\theta \\ 0 & \cos\theta & -\sin\theta \end{bmatrix} \cdot \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos\theta & \sin\theta \\ 0 & -\sin\theta & \cos\theta \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & -1 \\ 0 & 1 & 0 \end{bmatrix} = S(i)$$

Thus;  $\frac{d\text{Rot}(x, \theta)}{d\theta} = S(i) \cdot \text{Rot}(x, \theta)$

$$\frac{d\text{Rot}(y, \theta)}{d\theta} = S(j) \cdot \text{Rot}(y, \theta)$$

$$\frac{d\text{Rot}(z, \theta)}{d\theta} = S(k) \cdot \text{Rot}(z, \theta)$$



Suppose that a rotation matrix  $R$  is time varying so that  $R = R(t)$  for every  $t \in \mathbb{R}$ .

$$\dot{R}(t) = S(t) \cdot R(t)$$

Example: Suppose that  $R(t) = \text{Rot}(x, \theta(t))$

$$\dot{R}(t) = \frac{dR}{dt} = \frac{dR}{d\theta} \cdot \frac{d\theta}{dt} = \dot{\theta} \cdot S(i) \cdot R(t) = S(\omega(t)) \cdot R(t)$$

where  $\omega(t) = i \cdot \dot{\theta}$  angular velocity about the x-axis.

Suppose  $P_1$  is a vector fixed in a coordinate system  $(O_1, x_1, y_1, z_1)$  and the frame is rotating relative to the frame  $(O, x_0, y_0, z_0)$

$$P_0 = R(t) \cdot P_1$$

velocity of  $P_0$  is  $\dot{P}_0$ :

$$\dot{P}_0 = S(\omega) \cdot R(t) \cdot P_1$$

$$\dot{P}_0 = S(\omega) \cdot P_0 = \omega \times P_0$$

Suppose that the homogenous transformation relating the above two frames is time dependent so that

$$H(t) = \begin{bmatrix} R(t) & d(t) \\ \underbrace{0}_{\text{zero row}} & \underbrace{1}_{1 \times 3} \end{bmatrix}$$

→ Rotational part  
→ translational part

$$P_0 = R(t) \cdot P_1 + d(t)$$

$$\dot{P}_0 = \dot{R}(t) \cdot P_1 + \dot{d}(t) = S(\underline{\omega}) \cdot R(t) \cdot P_1 + \dot{d}(t) = \underline{\omega} \times (R P_1) + \dot{d}(t)$$

$r \triangleq R P_1$  the vector from  $O_1$  to  $P_0$  expressed in  $(O_0 x_0 y_0 z_0)$ , the orientation of frame.

$v \triangleq \dot{d}(t) + \underbrace{\dot{R}(t) \cdot P_1}_{\text{rate of change of } P_1 \text{ expressed in the frame } (O_0 x_0 y_0 z_0)}$  the rate of which the origin  $O_1$  is moving.

If the vector  $P_1$  is also changing relative to the frame  $(O_1 x_1 y_1 z_1)$  the term  $R(t) \dot{P}_1$  is to be added to the term  $v$ .

$$v \triangleq \dot{d}(t) + R(t) \dot{P}_1 + R(t) \ddot{P}_1$$

For the acceleration, note that:

$$\frac{d(\underline{a} \times \underline{b})}{dt} = \frac{d\underline{a}}{dt} \times \underline{b} + \underline{a} \times \frac{d\underline{b}}{dt}$$

$$\dot{P}_0 = \dot{R} P_1 + \dot{d} = (\underline{\omega} \times R P_1) + \dot{d}$$

$$\ddot{P}_0 = \dot{\underline{\omega}} \times R P_1 + \underline{\omega} \times (\dot{R} P_1) + \ddot{d}$$

$$= \underbrace{\dot{\underline{\omega}} \times r}_{\text{Transverse acceleration}} + \underbrace{\underline{\omega} \times (\underline{\omega} \times r)}_{\text{Centripetal acceleration}} + \ddot{d}$$

linear acceleration

of the particle and it is always directed toward the axis of rotation and is  $\perp$  to the axis.

If  $P_1$  is changing w.r. to  $(O_1 x_1 y_1 z_1)$  then the last eqn becomes;

$$\ddot{P}_0 = \dot{\underline{\omega}} \times r + \underline{\omega} \times (\underline{\omega} \times r) + \underbrace{2 \underline{\omega} \times R \dot{P}_1}_{\text{Coriolis acceleration}} + \ddot{d}$$

$$a = R \ddot{P}_1 + \ddot{d}$$

Addition of Angular Velocities:

$${}^0P_1 = {}^0R_1 \cdot P_1 + {}^0d_1$$

$$P_1 = {}^1R_2 \cdot P_2 + {}^1d_2$$

$$P_0 = {}^0R_2 \cdot P_2 + {}^0d_2$$

$${}^0R_2 = {}^0R_1 \cdot {}^1R_2$$

$${}^0d_2 = {}^0d_1 + {}^0R_1 \cdot {}^1d_2$$

$${}^0\dot{R}_2 = {}^0\dot{R}_1 \cdot {}^1R_2 + {}^0R_1 \cdot \dot{{}^1R}_2$$

$${}^0\dot{R}_2 = S({}^0\omega_2) \cdot {}^0R_2$$

$${}^0\dot{R}_1 \cdot {}^1R_2 = S({}^0\omega_1) \cdot {}^0R_1 \cdot {}^1R_2 = S(\omega_1) \cdot {}^0R_2$$

$$\begin{aligned} {}^0R_1 \cdot \dot{{}^1R}_2 &= {}^0R_1 \cdot S({}^1\omega_2) \cdot {}^1R_2 = {}^0R_1 \cdot S({}^1\omega_2) \cdot {}^0R_1^T \cdot {}^0R_1 \cdot {}^1R_2 \\ &= S({}^0R_1 \cdot {}^1\omega_2) \cdot {}^0R_1 \cdot {}^1R_2 = S({}^0R_1 \cdot {}^1\omega_2) \cdot {}^0R_2 \end{aligned}$$

$$S({}^0\omega_2) \cdot {}^0R_2 = [S({}^0\omega_1) + S({}^0R_1 \cdot {}^1\omega_2)] \cdot {}^0R_2$$

$$S(a) + S(b) = S(a+b)$$

$$S({}^0\omega_2) \cdot {}^0R_2 = S({}^0\omega_1 + {}^0R_1 \cdot {}^1\omega_2) \cdot {}^0R_2$$

$${}^0\omega_2 = {}^0\omega_1 + {}^0R_1 \cdot {}^1\omega_2$$

Generalization to n many frames,

$${}^0R_n = {}^0R_1 \cdot {}^1R_2 \cdots {}^{n-1}R_n$$

$${}^0\dot{R}_n = S({}^0\omega_n) \cdot {}^0R_n$$

$$\boxed{{}^0\omega_n = {}^0\omega_1 + {}^0R_1 \cdot {}^1\omega_2 + {}^0R_2 \cdot {}^2\omega_3 + \cdots + {}^0R_{n-1} \cdot {}^{n-1}\omega_n}$$

Inverse velocity and acceleration:

Consider an n-link manipulator with joint variables  $q_1, \dots, q_n$ . Let

$${}^0T_n(q) = \begin{bmatrix} {}^0R_n(q) & | & {}^0d_n(q) \\ \hline \underline{0} & & \underline{1} \end{bmatrix}$$

$$S({}^0\omega_n) = {}^0\dot{R}_n \cdot ({}^0R_n)^T$$

$${}^0v_n = {}^0\dot{d}_n$$

$${}^0v_n = J_v \cdot \dot{q}$$

$${}^0\omega_n = J_w \cdot \dot{q}$$

$$\begin{bmatrix} {}^0v_n \\ \hline {}^0\omega_n \end{bmatrix} = {}^0J_n \cdot \dot{q}$$

$$\Rightarrow {}^0J_n = \begin{bmatrix} J_v \\ \hline J_w \end{bmatrix} \quad \text{Manipulator Jacobian}$$

$$\begin{bmatrix} dx/dt \\ dy/dt \\ dz/dt \\ \hline \delta x/dt \\ \delta y/dt \\ \delta z/dt \end{bmatrix} = \begin{bmatrix} J \\ \hline J \end{bmatrix} \begin{bmatrix} dq_1/dt \\ dq_2/dt \\ \vdots \\ dq_n/dt \end{bmatrix}$$

Recall that angular velocities can be added vectorially provided that they're expressed relative to a common coordinate frame. Thus the angular velocity of end-effector relative to the base by expressing the angular velocity of the each link in the orientation of the base frame and then summing them.

If the  $i^{\text{th}}$  joint is revolute then the  $i^{\text{th}}$  joint variable  $q_i$  equals  $\theta_i$  and the axis of rotation is  $z_{i-1}$  axis. Thus the angular velocity of link  $i$  expressed in the frame  $i-1$  is given by

$${}^{i-1}\omega_i = \dot{q}_i \cdot \underbrace{k}_{\substack{\leftarrow \text{unit vector in frame } i-1 \\ \downarrow \\ \hat{z}_i}}$$

If the  $i^{\text{th}}$  joint is prismatic then the motion of frame  $i$  relative to frame  $i-1$  is a translation and  ${}^{i-1}\omega_i = 0$ . Since  $q_i = d_i$ , now there is no rotation.

The overall angular velocity of the end-effector  ${}^0\omega_n$  in the base frame is then

$${}^0\omega_n = \beta_1 \dot{q}_1 \cdot k + \beta_2 \dot{q}_2 \cdot {}^0R_1 \cdot k + \dots + \beta_n \dot{q}_n \cdot R_{n-1} \cdot k$$

$${}^0\omega_n = \sum_{i=1}^n \beta_i \dot{q}_i \cdot z_{i-1}$$

where  $z_{i-1} = {}^0R_{i-1} \cdot k$  and  $\beta_i = 1$  if the joint is revolute; and equal to 0 if the joint is prismatic.

Note that,  $z_0 = k = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$ . The lower part of Jacobian,  $J_w$ , is thus given by

$$J_w = [\beta_1 z_0 \quad \beta_2 z_1 \quad \dots \quad \beta_n z_{n-1}]$$

The linear velocity of the end-effector is just  ${}^0\dot{d}_n$ . By the chain rule for differentiation

$${}^0\dot{d}_n = \sum_{i=1}^n \frac{\partial \dot{d}_n}{\partial q_i} \cdot \dot{q}_i$$

Thus the  $i^{\text{th}}$  column of  $J_v$  is just  $\frac{\partial \dot{d}_n}{\partial q_i}$ . Furthermore, this expression is just the linear velocity of the end-effector that would result if  $\dot{q}_i$  is equal to one and the other  $\dot{q}_j$  are zero. In other words, the  $i^{\text{th}}$  column of the Jacobian is generated by holding all joints fixed but the  $i^{\text{th}}$  and actuating the  $i^{\text{th}}$  at the unit velocity

Inverse velocity and acceleration:

$$\begin{aligned} \dot{v}_n &= J_v \dot{q} & {}^0 J_n &= \begin{bmatrix} -J_v \\ J_w \end{bmatrix} \\ \dot{w}_n &= J_w \dot{q} \end{aligned}$$

$${}^0 \dot{w}_n = \rho_1 \dot{q}_1 {}^k P_2 \dot{q}_2 {}^0 R_{1+k} + \dots + \rho_n \dot{q}_n {}^0 R_{n-k} = \sum_{i=1}^n \rho_i \dot{q}_i z_{i-1}$$

$$z_{i-1} = {}^0 R_{i-1} k$$

$\rho_i = 1$  if the joint is revolute.  
 $= 0$  if the joint is prismatic.

$$J_w = \begin{bmatrix} \rho_1 z_0 & \rho_2 z_1 & \dots & \rho_n z_{n-1} \end{bmatrix}$$

$${}^0 \dot{d}_n = \sum_{i=1}^n \frac{\partial d_n}{\partial q_i} \dot{q}_i$$

$$\rho_n \dot{q}_n {}^0 R_{n-k}$$

J is just



$$\frac{\partial d_n}{\partial q_i}$$

${}^0 \dot{d}_n$  is just the linear velocity of the end-effector that would result if  $\dot{q}_i$  is equal to "1" and the other  $\dot{q}_j$  are "0".

Thus the  $i$ -th column of  ${}^0 J_n$  is generated by holding all joints fixed but the  $i$ -th, and activating the  $i$ -th at unit velocity. Consider these two cases separately.

Case 1

If joint  $i$  is prismatic, then  ${}^0 R_{j-1}$  is independent of  $q_i = d_i$  for all  $j$  and

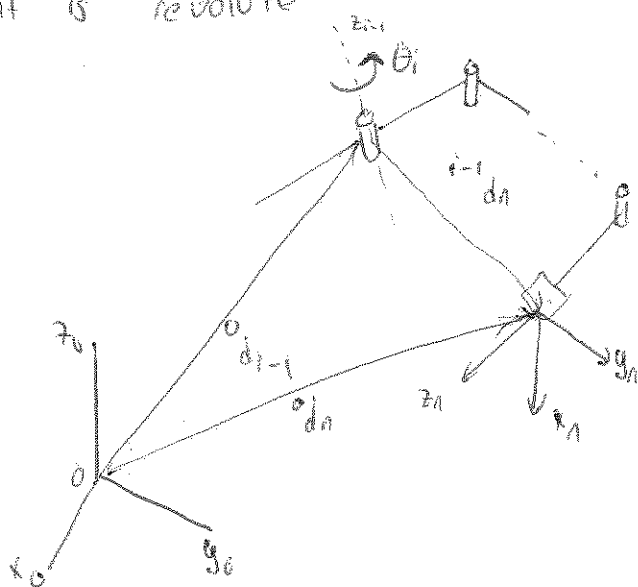
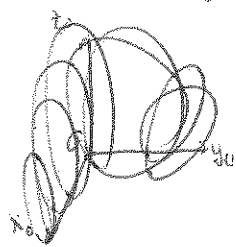
$$\boxed{{}^{i-1} d_i = d_i + {}^{i-1} R_i \partial_i i}$$

If all joints are fixed but the  $i$ -th we have that

$${}^0 \dot{d}_n = {}^0 R_{i-1} \dot{d}_i = \dot{d}_i {}^0 R_{i-1} k = \dot{d}_i z_{i-1} \rightarrow \frac{\partial d_n}{\partial q_i} = z_{i-1}$$

Case 2

If the joint is revolute



Vector  ${}^0d_k$  from the origin  $O_0$  to the origin  $O_k$  for any  $k$  and write

$$* {}^0d_n = {}^0d_{i-1} + {}^0R_{i-1} {}^{i-1}d_n$$

or in the new notation

$$* {}^0o_n - {}^0o_{i-1} = {}^0R_{i-1} {}^{i-1}d_n$$

Referring to the scheme we consider, note that both  ${}^0d_{i-1}$  and  ${}^0R_{i-1}$  are constant if only the  $i$ -th joint is actuated.

Therefore

$${}^0\dot{d}_n = {}^0R_{i-1} {}^{i-1}\dot{d}_n$$

Since the motion of link  $i$  is a rotation  $q_i$  about  $z_{i-1}$  we have

$${}^{i-1}\dot{d}_n = \dot{q}_i k \times {}^{i-1}d_n$$

and thus

$${}^0\dot{d}_n = {}^0R_{i-1} (\dot{q}_i k \times {}^{i-1}d_n) = \dot{q}_i {}^0R_{i-1} k \times {}^0R_{i-1} {}^{i-1}d_n = \dot{q}_i z_{i-1} \times ({}^0o_n - {}^0o_{i-1})$$

Hence

$$\frac{\partial {}^0d_n}{\partial q_i} = z_{i-1} \times ({}^0o_n - {}^0o_{i-1})$$



$$J_v = [J_{v_1} \dots J_{v_n}]$$

$J_{v_i} = z_{i-1} \times (o_i - o_{i-1}) \rightarrow$  If joint is revolute

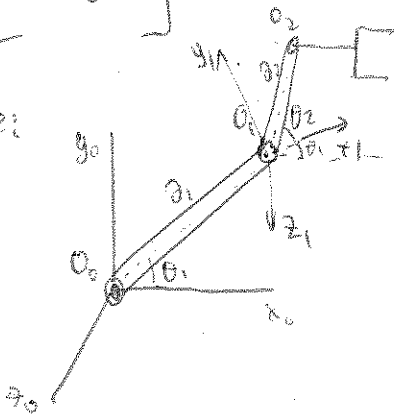
$J_{v_i} = z_{i-1} \rightarrow$  prismatic

$$J = [J_1, J_2, \dots, J_n]$$

$$J_i = \begin{bmatrix} z_{i-1} \times (o_i - o_{i-1}) \\ z_{i-1} \end{bmatrix} \text{ for revolute}$$

$$= \begin{bmatrix} z_{i-1} \\ 0 \end{bmatrix} \text{ for prismatic}$$

Example:



Both joints are revolute.

The Jacobian in this case is of the form

$$J(q) = \begin{bmatrix} z_0 \times (o_2 - o_0) & z_1 \times (o_2 - o_1) \\ z_0 & z_1 \end{bmatrix}$$

$$o_0 = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} \quad o_1 = \begin{bmatrix} a_1 c_1 \\ a_1 s_1 \\ 0 \end{bmatrix}$$

$$o_2 = \begin{bmatrix} a_1 c_1 + a_2 c_2 \\ a_1 s_1 + a_2 s_2 \\ 0 \end{bmatrix}$$

$z_0 = 0, 1, 0, 1, 1, 1$   
 $z_1 = 0, 1, 0, 1, 1, 1$

$$o_2 - o_0 = o_2 \quad o_2 - o_1 = \begin{bmatrix} a_2 c_2 \\ a_2 s_2 \\ 0 \end{bmatrix}$$

$$J = \begin{bmatrix} (-a_2 s_2 - a_1 s_1) & -a_2 c_2 \\ (a_1 c_1 + a_2 c_2) & a_2 s_2 \\ 0 & 0 \\ 0 & 0 \\ 0 & 0 \\ 1 & 1 \end{bmatrix}$$

$$[d\tau] = J [dq]$$

$$\begin{bmatrix} i & j & k \\ 0 & 0 & 1 \\ a_{11} & a_{12} & 0 \end{bmatrix} = -i a_{21} s_2 - j$$

|          |          |   |
|----------|----------|---|
| 1        | j        | 1 |
| 0        | 0        | 1 |
| $a_{11}$ | $a_{12}$ | 0 |

$$\begin{bmatrix} T_d \\ T_8 \end{bmatrix} = \begin{bmatrix} n_x & n_y & n_z & (p_{x1})_x & (p_{x1})_y & (p_{x1})_z \\ o_x & o_y & o_z & (p_{x0})_x & (p_{x0})_y & (p_{x0})_z \\ d_x & d_y & d_z & (p_{x0})_x & (p_{x0})_y & (p_{x0})_z \\ 0 & 0 & 0 & n_x & n_y & n_z \\ 0 & 0 & 0 & o_x & o_y & o_z \\ 0 & 0 & 0 & d_x & d_y & d_z \end{bmatrix} \begin{bmatrix} d \\ o_8 \end{bmatrix} \quad \text{with } (p_{x0})_i = d_i$$

Example: STANFORD MANIPULATOR

Joint 3 is prismatic  $\alpha_3 = \alpha_4 = \alpha_5 = 0$

as a consequence of the spherical wrist and the coordinate frame assignment.

$$J = \begin{bmatrix} z_0 \times (o_3 - o_0) & z_1 \times (o_4 - o_1) & z_2 & z_3 \times (o_6 - o_3) & z_4 \times (o_6 - o_4) & z_5 \times (o_6 - o_5) \\ z_0 & z_1 & 0 & z_3 & z_4 & z_5 \end{bmatrix}$$

Construct A matrices

- Any  $o_j$  is given by the first three entries of the last column, of

$${}^0T_j = A_1 \dots A_j \quad \text{with } o_0 = \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix} = o_1$$

- Any  $z_j$  is given by

$$z_j = {}^0R_{jk}$$

↑ Rotational part of  ${}^0T_j$

Thus it is only necessary to compute the matrices  ${}^0T_j$  to calculate the Jacobian.

Thus,

$$o_6 = \begin{bmatrix} c_1 s_2 d_3 - s_1 d_2 + d_6 (c_1 c_2 c_3 s_5 + c_1 c_3 s_2 - s_1 s_4 s_5) \\ s_1 s_2 d_3 - s_1 d_2 + d_6 (c_1 s_4 s_5 + c_1 c_4 s_1 s_2 + c_5 s_1 s_2) \\ c_2 d_3 + d_6 (c_2 (s_5 - c_4 s_2 s_5)) \end{bmatrix}$$

$$a_3 = \begin{bmatrix} c_2 d_3 - s_1 d_2 \\ s_1 d_2 + c_1 d_2 \\ c_2 d_3 \end{bmatrix} = 0 \rightarrow \text{not zero}$$

and

$$z_0 = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \quad z_1 = \begin{bmatrix} -s_1 \\ c_1 \\ 0 \end{bmatrix} \quad z_2 = \begin{bmatrix} c_1 s_2 \\ s_1 s_2 \\ c_2 \end{bmatrix} \quad z_3 = \begin{bmatrix} c_1 s_2 \\ s_1 s_2 \\ c_2 \end{bmatrix} \quad z_4 = \begin{bmatrix} -c_1 c_2 s_4 - s_1 c_4 \\ -s_1 c_2 s_4 + c_1 c_4 \\ s_2 s_4 \end{bmatrix}$$

$$z_5 = \begin{bmatrix} c_1 c_2 s_4 s_5 - s_1 s_2 s_5 + c_1 s_2 c_5 \\ s_1 c_2 c_4 s_5 + c_1 s_2 s_5 + s_1 s_2 c_5 \\ -s_1 c_2 s_5 + c_1 c_2 c_5 \end{bmatrix}$$

Returning back to joint velocities and end-effector velocities

$$\overset{\text{end effector velocity}}{\dot{x}} = J(q) \overset{\text{joint velocity}}{\dot{q}}$$

$$\ddot{x} = J(q) \ddot{q} + \left( \frac{dJ(q)}{dt} \right) \dot{q}$$

Given a vector  $\ddot{x}$  of end-effector accelerations, the instantaneous joint acceleration vector  $\ddot{q}$  is given as a solution of

$$b = J(q) \ddot{q}$$

where

$$b = \ddot{x} - \left( \frac{dJ(q)}{dt} \right) \dot{q} \quad \dot{q} = J(q)^{-1} \dot{x}$$

$$\ddot{q} = J(q)^{-1} b \quad \text{provided that } \det(J(q)) \neq 0$$

STATIC FORCES

$$F_2 = \begin{bmatrix} f_x \\ f_y \\ f_z \\ m_x \\ m_y \\ m_z \end{bmatrix}$$

$$f = 10i + 0j - 150k$$

$$m = 0i - 100j + 0k$$

## Transformation of forces between coordinate frames

- Given a force and moment acting at the origin of some coordinate frame attached to a fixed object.

- find the equivalent force and moment acting at, and describe w.r.t some other coordinate frame also attached rigidly to the object.

Use method of virtual work in the process to solve the problem.

Object is displaced by an imaginary differential amount  $D$  from the original position. This is known as a virtual displacement. During virtual displacement a virtual work  $\delta W$  is done.

$D$  is infinitesimally small so that the energy stored in the system does not change. Thus the  $\delta W$  done by a number of forces acting on the object is  $\delta W$ .

$$\text{Thus } \delta W = F^T D$$

$$D = [dx \quad dy \quad dz \quad \delta_x \quad \delta_y \quad \delta_z]^T$$

Suppose that this displacement  $D$  can be achieved by the application of another force and moment  ${}^C F$  acting at some different point on the object, described by a coordinate frame  $C$ . Then the same virtual work will result.

$$\delta W = {}^C F^T {}^C D$$

$$F^T D = {}^C F^T {}^C D$$

$$\text{define } {}^C D = J D$$

$$\begin{bmatrix} {}^C d_x \\ {}^C d_y \\ {}^C d_z \\ {}^C \delta_x \\ {}^C \delta_y \\ {}^C \delta_z \end{bmatrix} = \begin{bmatrix} n_x & n_y & n_z & (p_{xx})_x & - & - & - \\ 0_x & - & - & (p_{xx})_y & - & - & - \\ d_x & - & - & (p_{xx})_z & - & - & - \\ 0 & 0 & 0 & n_x & n_y & n_z & \\ 0 & 0 & 0 & \vdots & \vdots & \vdots & \\ 0 & 0 & 0 & \vdots & \vdots & \vdots & \end{bmatrix} \begin{bmatrix} dx \\ dy \\ dz \\ \delta_x \\ \delta_y \\ \delta_z \end{bmatrix}$$

$$F^T D = {}^c F^T J D$$

$$F^T = {}^c F^T J$$

$$F = J^T {}^c F$$

$$\begin{bmatrix} f_x \\ f_y \\ f_z \\ m_x \\ m_y \\ m_z \end{bmatrix} = \begin{bmatrix} c_{fx} \\ c_{fy} \\ c_{fz} \\ c_{mx} \\ c_{my} \\ c_{mz} \end{bmatrix}$$

$$= \begin{bmatrix} n_x & n_y & n_z \\ o_x & o_y & o_z \\ d_x & d_y & d_z \end{bmatrix} \begin{bmatrix} n_x \\ n_y \\ n_z \\ o_x \\ o_y \\ o_z \\ d_x \\ d_y \\ d_z \end{bmatrix}$$

$${}^c F = (J^T)^{-1} F$$

$$= \begin{bmatrix} n_x & n_y & n_z & 0 & 0 & 0 \\ o_x & o_y & o_z & 0 & 0 & 0 \\ d_x & d_y & d_z & 0 & 0 & 0 \\ (p_{rx})_x & (p_{rx})_y & (p_{rx})_z & n_x & n_y & n_z \\ (p_{ry})_x & (p_{ry})_y & (p_{ry})_z & o_x & o_y & o_z \\ (p_{rz})_x & (p_{rz})_y & (p_{rz})_z & d_x & d_y & d_z \end{bmatrix} \begin{bmatrix} f_x \\ f_y \\ f_z \\ m_x \\ m_y \\ m_z \end{bmatrix}$$

$$\begin{bmatrix} c_{mx} \\ c_{my} \\ c_{mz} \\ c_{fx} \\ c_{fy} \\ c_{fz} \end{bmatrix} = \begin{bmatrix} m_x \\ m_y \\ m_z \\ f_x \\ f_y \\ f_z \end{bmatrix} J$$

$$\begin{bmatrix} C_{m_x} \\ C_{m_y} \\ C_{m_z} \\ C_{f_x} \\ C_{f_y} \\ C_{f_z} \end{bmatrix} = J \begin{bmatrix} m_x \\ m_y \\ m_z \\ f_x \\ f_y \\ f_z \end{bmatrix}$$

$$C_{m_x} = n((f \times p) + m)$$

$$C_{m_y} = o((f \times p) + m)$$

$$C_{m_z} = d((f \times p) + m)$$

$$C_{f_x} = n \cdot f$$

$$C_{f_y} = o \cdot f$$

$$C_{f_z} = d \cdot f$$

### Example:

Given the coordinate frame

$$A = \begin{bmatrix} 0 & 0 & 1 & 10 \\ 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

and the following force and moment applied in the base coordinates

$$f = 10i + 0j + 0k$$

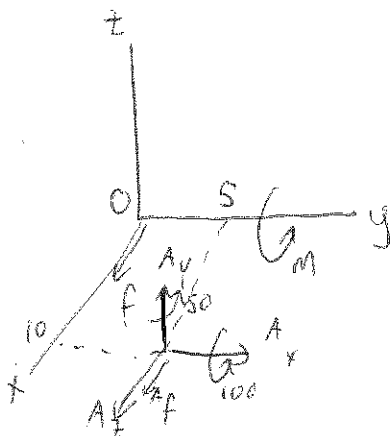
$$m = 0i + 100j + 0k$$

What is the equivalent force and moment in coordinate frame A?

$${}^A M_x = n((f \times p) + m)$$

$$f \times p = \begin{vmatrix} i & j & k \\ 10 & 0 & 0 \\ 0 & 5 & 0 \end{vmatrix} = 50k$$

$$(f \times p) + m = 100j + 50k$$



$${}^A M_x = [0 \ 1 \ 0] \begin{bmatrix} 0 \\ 100 \\ 50 \end{bmatrix} = 100$$

$${}^A M_y = [0 \ 0 \ 1] \begin{bmatrix} 0 \\ 100 \\ 50 \end{bmatrix} = 50$$

$${}^A M_z = [1 \ 0 \ 0] \begin{bmatrix} 0 \\ 100 \\ 50 \end{bmatrix} = 0$$

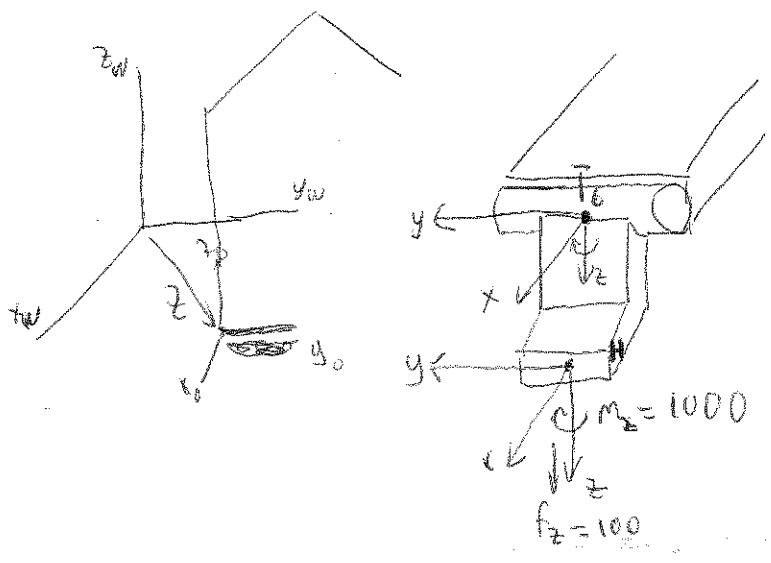
$${}^A M = 100i + 50j + 0k$$

$${}^A f_x = n \cdot f = 0 \quad {}^A f = 0i + 0j + 10k$$

$${}^A f_y = o \cdot f = 0$$

$${}^A f_z = \partial \cdot f = 10$$

Example: A manipulator and end-effector are positioned by  ${}^z T_6 E$  to insert a screw into a hole described by  $O, H$  as shown below.



The end-effector is described by

$$E = \begin{bmatrix} 1 & 0 & 0 & 2 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 10 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

The manipulator is to exert a force of

$$f = 0i + 0j + 100k$$

and a moment of

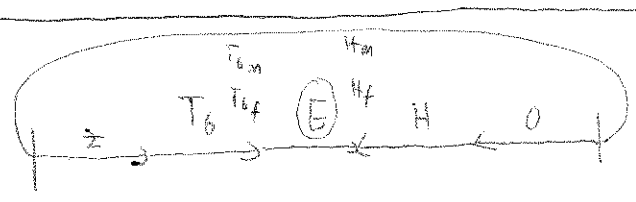
$$m = 0i + 0j + 1000k$$

$${}^z T_6 E = O, H$$

$${}^z T_6 E E^{-1} = O, H E^{-1}$$

$${}^z T_6 = O, H E^{-1}$$

in hole coordinates,  $H$ . What is the equivalent force and moment in  $T_6$ ?



$$E^{-1} = \begin{bmatrix} 1 & 0 & 0 & -2 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & -10 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$p = -2i + 0j - 10k$$

$$f \times p = \begin{vmatrix} i & j & k \\ 0 & 0 & 100 \\ -2 & 0 & -10 \end{vmatrix} = -200j$$

$$(f + p) \cdot m = 200j + 1000k$$

$$T_{0m_x} = n \cdot ((f \times p) + m) = [1 \ 0 \ 0] \begin{bmatrix} 0 \\ -200 \\ 1000 \end{bmatrix} = 0$$

$$T_{0f_x} = n \cdot f = 0$$

$$T_{0m_y} = o \cdot ((f \times p) + m) = [0 \ 1 \ 0] \begin{bmatrix} 0 \\ -200 \\ 1000 \end{bmatrix} = -200$$

$$T_{0f_y} = o \cdot f = 0$$

$$T_{0m_z} = \partial \cdot ((f \times p) + m) = [0 \ 0 \ 1] \begin{bmatrix} 0 \\ -200 \\ 1000 \end{bmatrix} = 1000$$

$$T_{0f_z} = \partial \cdot f = 100$$

$$T_{0m} = 0i - 200j + 1000k$$

$$T_{0f} = 0i + 0j + 100k$$

Forces, Moments, and Equivalent Joint Torques

$$SW = {}^{T_0}F^T {}^{T_0}D = Z^T Q = [z_1 \ z_2 \ z_3 \ z_4 \ z_5 \ z_6] \begin{bmatrix} d\theta_1 \\ d\theta_2 \\ d\theta_3 \\ d\theta_4 \\ d\theta_5 \\ d\theta_6 \end{bmatrix}$$

$$\begin{bmatrix} dx \\ dy \\ dz \\ \delta_x \\ \delta_y \\ \delta_z \end{bmatrix}$$

$${}^{T_0}D = JQ$$

$${}^{T_0}F^T JQ = Z^T Q$$

$${}^{T_0}F^T J = Z^T$$

$$\boxed{Z = J^T {}^{T_0}F \leftarrow \text{Given}}$$

equivalent joint Torques



Example: A Stanford manipulator is in a state such that

$$T_6 = \begin{bmatrix} 0 & 1 & 0 & 20 \\ 1 & 0 & 0 & 6 \\ 0 & 0 & -1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

which corresponds to the following joint coordinates whose sine and cosine entries are as follows,

Manipulator State

| Coordinate | Value      | $S_i$ | $C_i$ |
|------------|------------|-------|-------|
| $\theta_1$ | 0          | 0     | 1     |
| $\theta_2$ | $90^\circ$ | 1     | 0     |
| $d_3$      | $20''$     |       |       |
| $\theta_4$ | 0          | 0     | 1     |
| $\theta_5$ | $90^\circ$ | 1     | 0     |
| $\theta_6$ | $90^\circ$ | 1     | 0     |

$$\frac{\partial T_6}{\partial q_i} = \begin{bmatrix} \frac{\partial T_6}{\partial \theta_1} & \frac{\partial T_6}{\partial \theta_2} & \frac{\partial T_6}{\partial d_3} & \frac{\partial T_6}{\partial \theta_4} & \frac{\partial T_6}{\partial \theta_5} & \frac{\partial T_6}{\partial \theta_6} \end{bmatrix}$$

$$\frac{\partial T_6}{\partial \theta_6} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 1 \end{bmatrix} \quad \frac{\partial T_6}{\partial \theta_5} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ S_6 \\ C_6 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 1 \\ 0 \\ 0 \end{bmatrix} \quad \frac{\partial T_6}{\partial \theta_4} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ -S_5 C_6 \\ S_5 S_6 \\ C_5 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ 1 \\ 0 \end{bmatrix}$$

$$\frac{\partial T_6}{\partial d_3} = \begin{bmatrix} -S_5 C_6 \\ S_5 S_6 \\ C_5 \\ 0 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 \\ 1 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} \quad \frac{\partial T_6}{\partial \theta_2} = \begin{bmatrix} d_3 (C_4 C_5 C_6 - S_4 S_6) \\ -d_3 (C_4 C_5 C_6 + S_4 S_6) \\ d_3 C_4 S_5 \\ S_4 C_5 C_6 + C_4 S_6 \\ -S_4 C_5 S_6 + C_4 C_6 \\ S_4 S_5 \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 20 \\ 1 \\ 0 \\ 0 \end{bmatrix}$$

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$$\frac{dT_0}{d\theta_i} = \begin{bmatrix} -d_2 [c_2 (c_4 c_5 c_6 - s_4 s_6) - s_2 s_5 c_6] + s_2 d_3 (s_4 c_5 c_6 + c_4 s_6) \\ -d_2 [-c_2 (c_4 c_5 s_6 + s_4 c_6) + s_2 s_5 s_6] + s_2 d_3 (-s_4 c_5 s_6 + c_4 c_6) \\ -d_2 (c_2 c_4 s_5 + s_2 c_5) + s_2 d_3 s_4 s_5 \\ -s_2 (c_4 c_5 c_6 - s_4 s_6) - c_2 s_5 c_6 \\ s_2 (c_4 c_5 s_6 + s_4 c_6) + c_2 s_5 s_6 \\ -s_2 c_4 c_5 + c_2 c_5 \end{bmatrix} = \begin{bmatrix} 20 \\ -6 \\ 0 \\ 0 \\ 0 \\ -1 \end{bmatrix}$$

$$\frac{\partial T_0}{\partial \theta_i} = \begin{bmatrix} 20 & 0 & 0 & 0 & 0 & 0 \\ -6 & 0 & 1 & 0 & 0 & 0 \\ 0 & 20 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ -1 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} = J$$

We are to compute joint torques and forces necessary to exert the force and moment

$$\vec{T}_0 = \vec{F} = \begin{bmatrix} 0 \\ 0 \\ 100 \\ 0 \\ -200 \\ 1000 \end{bmatrix}$$

$$\frac{\partial \bar{T}_6}{\partial \dot{q}_i} = J = \begin{bmatrix} 20 & 0 & 0 & 0 & 0 & 0 \\ -6 & 0 & 1 & 0 & 0 & 0 \\ 0 & 20 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ -1 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}$$

$$\bar{T}_6 f = \begin{bmatrix} 0 \\ 0 \\ 100 \\ 0 \\ -200 \\ 1000 \end{bmatrix}$$

*u*

$$Z = J^T b f$$

$$Z = J^T \bar{T}_6 f$$

$$\begin{bmatrix} z_1 \\ z_2 \\ z_3 \\ z_4 \\ z_5 \\ z_6 \end{bmatrix} = \begin{bmatrix} J^T \\ \vdots \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 100 \\ 0 \\ -200 \\ 1000 \end{bmatrix} = \begin{bmatrix} -1000 \\ 2000 \\ 0 \\ -200 \\ 0 \\ 1000 \end{bmatrix}$$

## CONTROL OF MANIPULATORS

$$Z_i = \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = \sum_{j=1}^n \sum_{k=1}^6 \text{tr} \left\{ \frac{\partial T_j}{\partial q_k} J_j \frac{\partial T_j^T}{\partial q_i} \right\} \ddot{q}_k + \sum_{j=1}^n \sum_{k=1}^6 \sum_{l=1}^6 \text{tr} \left\{ \frac{\partial T_j}{\partial q_k} J_j \frac{\partial T_j^T}{\partial q_l} \frac{\partial T_j^T}{\partial q_i} \right\} \dot{q}_k \dot{q}_l$$

$$= \sum_{j=1}^n m_j g_j^T \frac{\partial T_j}{\partial q_i} \begin{matrix} j \\ r \\ j \end{matrix} \quad \text{for } i=1, 2, \dots, n$$

Pseudo inertia matrix of link  $i$  w.r.t. the  $i$ -th coordinate frame

position of the center of mass of link  $i$  w.r.t. the  $j$ -th coordinate frame

$$= \sum_{k=1}^n D_{ik} \ddot{q}_k + \sum_{k=1}^n \sum_{m=1}^n B_{ikm} \dot{q}_k \dot{q}_m + C_i$$

$$Z(t) = D(q(t)) \ddot{q}(t) + h(q(t), \dot{q}(t)) + C(q(t))$$

$D(q(t))$   $n \times n$  inertial acceleration-related symmetric matrix, with elements

$$D_{ik} = \sum_{j=\max(i,k)}^n \text{tr} \left( \frac{\partial T_j}{\partial q_k} J_j \frac{\partial T_j^T}{\partial q_i} \right) \quad i, k = 1, 2, \dots, n$$

$h(q, \dot{q})$   $n \times 1$  Coriolis and centrifugal force vector whose elements are

$$h(q, \dot{q}) = (h_1, h_2, \dots, h_n)^T$$

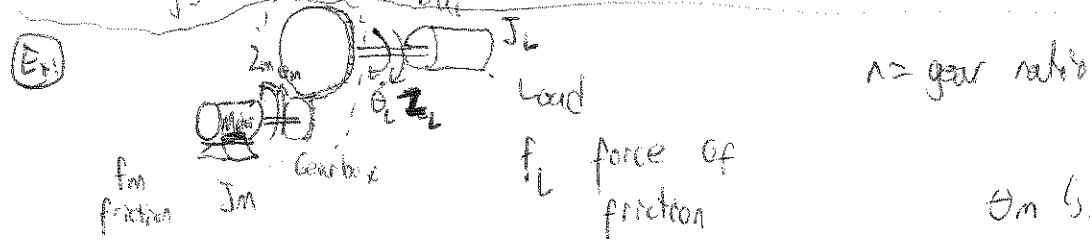
$$h_i = \sum_{k=1}^n \sum_{m=1}^n h_{ikm} \dot{q}_k \dot{q}_m \quad i = 1, 2, \dots, n$$

$$h_{ikm} = \sum_{j=\max(i,k,m)}^n \text{tr} \left( \frac{\partial T_j}{\partial q_k} \frac{\partial T_j}{\partial q_m} J_j \frac{\partial T_j^T}{\partial q_i} \right)$$

$c(q)$   $n \times 1$  gravity loading force vector

$$c(q) = (c_1, c_2, \dots, c_n)^T$$

$$c_i = \sum_{j=1}^n (-m_j g^T \frac{\partial T_j}{\partial q_i} j_{r_j}) \quad i = 1, 2, \dots, n$$



$$\theta_m(s) = \frac{\theta_L(s)}{n}$$

$$Z_L = J_L \ddot{\theta}_L(t) + f_L \dot{\theta}_L(t)$$

$$Z_M = J_M \ddot{\theta}_M(t) + f_M \dot{\theta}_M(t)$$

$$Z(t) = Z_M(t) + n^2 Z_L$$

$$Z(t) = (J_M + n^2 J_L) \ddot{\theta}_M(t) + (f_M + n^2 f_L) \dot{\theta}_M(t)$$

$$= J_{\text{eff}} \ddot{\theta}_M(t) + f_{\text{eff}} \dot{\theta}_M(t)$$

Effective inertia  $\uparrow$   $\leftarrow$  effective viscous friction

$$i_a(t) = K_a \dot{\theta}_m \quad \text{armature current}$$

$$v_a(t) = R_a i_a + L_a \frac{di_a}{dt} + e_g$$

armature  
resistance

armature  
inductance

back  
emf

$$e_g = K_g \dot{\theta}_m(t)$$

$$(R_a + L_a s) i_a(s) = s^2 J_{eff} \theta_m(s) + s f_{eff} \theta_m(s) + K_g s \theta_m(s) + K_a \dot{\theta}_m(s)$$

$$T(s) = \frac{\theta_m(s)}{v_a(s)} = \frac{s^2 J_{eff} \theta_m(s) + s f_{eff} \theta_m(s)}{R_a i_a(s)} = K_a \left[ \frac{v_a(s) - s K_g \theta_m(s)}{R_a L_a s} \right]$$

$$T_a = \frac{L_a}{R_a} \quad \text{electrical time constant of the motor armature}$$

$$T_m \gg T_a$$

$T_m$  = mechanical time constant of the motor

$$\frac{\theta_m(s)}{v_a(s)} = \frac{K_a}{s(s K_a J_{eff} + R_a f_{eff} + K_a K_g)} = \frac{K}{s(T_m s + 1)}$$

$$K = \frac{K_a}{R_a f_{eff} + K_a K_g} \quad \text{motor gain}$$

$$T_m = \frac{R_a J_{eff}}{R_a f_{eff} + K_a K_g} \quad \text{motor time constant}$$

$$\frac{\theta_m(s)}{V_a(s)} = \frac{K_a}{s(sR_a J_{eff} + R_a f_{eff} + K_a K_g)} = \frac{K}{s(T_m s + 1)}$$

$$K \triangleq \frac{K_a}{R_a f_{eff} + K_a K_g}$$

$$T_m \triangleq \frac{R_a J_{eff}}{R_a f_{eff} + K_a K_g}$$

$$\frac{\theta_L(s)}{V_a(s)} = \frac{nK}{s(T_m s + 1)}$$

Positional Control for a Single Joint

$$V_a(t) = \frac{K_p e(t)}{n} = \frac{K_p (\theta_L \text{ desired} - \theta_{\text{actual}})}{n}$$

$$V_a(s) = \frac{K_p E(s)}{n}$$

$$\frac{\theta_L(s)}{E(s)} \triangleq G(s) = \frac{K_a K_p}{s(sR_a J_{eff} + R_a f_{eff} + K_a K_g)}$$

$$\frac{\theta_{\text{act}}(s)}{\theta_{\text{des}}(s)} \triangleq \frac{E(s)}{1+G(s)} = \frac{K_a K_p / R_a J_{eff}}{s^2 + [(R_a J_{eff} + K_a K_g) / R_a J_{eff}] s + K_a K_g / R_a J_{eff}}$$

In order to increase the system response time and reduce the steady-state error, increase the positional feedback gain and add some damping into the system by adding a derivative of the positional error.

$$V_a(t) = \frac{K_p [\theta_{\text{des}} - \theta_{\text{actual}}] + K_v [\dot{\theta}_{\text{des}} - \dot{\theta}_{\text{act}}]}{n} = \frac{K_p e(t) + K_v \dot{e}(t)}{n}$$

$$\frac{\theta_L(s)}{E(s)} \triangleq G_{pd}(s) = \frac{K_a K_v s + K_a K_p}{s(sR_a J_{eff} + R_a f_{eff} + K_a K_g)}$$

$$\frac{\theta_{\text{act}}(s)}{\theta_{\text{des}}(s)} \triangleq \frac{G_{pd}(s)}{1+G_{pd}(s)} = \frac{K_a K_v s + K_a K_p}{s^2 R_a J_{eff} + s(R_a f_{eff} + K_a K_g + K_a K_v) + K_a K_p}$$

In order not to excite the structural oscillation and resonance of the joint, the undamped natural frequency  $\omega_n$  may be set to no more than 0.5 of the structural resonant frequency of the joint, that is

$$\omega_n \leq 0.5 \omega_r$$

$K_{stiff}$  : effective stiffness of the joint

$Z_{restoring}$  :  $K_{stiff} \theta_m(t) = Z_{inertia}$  of the motor

$$J_{eff} \ddot{\theta}_m(t) + K_{stiff} \theta_m(t) = 0$$

$$J_{eff} s^2 + K_{stiff} = 0$$

$$\omega_r = \sqrt{\frac{K_{stiff}}{J_{eff}}} \quad \leftarrow \text{is constant}$$

$$\omega_0 = \sqrt{\frac{K_{stiff}}{J_0}}$$

$$\omega_r = \omega_0 \sqrt{\frac{J_0}{J_{eff}}}$$

$$\omega_n \leq 0.5 \omega_r$$

$$0 < K_p < \frac{\omega_0^2 J_0 k_a}{4 k_a}$$

$$K_v \geq \frac{R_a \omega_0 \sqrt{J_0 / J_{eff}} - R_a f_{eff} - k_a k_g}{k_a}$$

Steady-State Errors of the system

$$E(s) = \theta_{L_{des}}(s) - \theta_{L_{act}}(s)$$

① For a step input of Magnitude  $A$

$$\theta_{L_{des}}(t) = A$$

$$e_{ss}(\text{step}) \triangleq e_{ssp} = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} s E(s) = \lim_{s \rightarrow 0} s \frac{[s^2 J_{eff} k_a (s R_a f_{eff} + k_a k_g)] \frac{A}{s}}{s^2 R_a J_{eff} (s (R_a f_{eff} + k_a k_g + k_a k_v) + k_a k_p)}$$

$$= \lim_{s \rightarrow 0} s \left[ \frac{(n R_a D(s))}{A} \right] \quad \Delta$$

$$Z_D(t) = Z_c(t) + Z_c(t) + Z_e$$

$$D(s) = \frac{7e}{s} + \frac{7e}{s} + \frac{7e}{s}$$

$$D(q)\ddot{q}(t) + h(q, \dot{q}) + C(q, \dot{q}) = D_a(q, \dot{q}) \left\{ \ddot{q}^d(t) + K_v [\dot{q}^d(t) - \dot{q}(t)] + K_p [q^d(t) - q(t)] \right\} + h_a(q, \dot{q}) + C(q)$$

if  $D_a(q)$ ,  $h_a(q, \dot{q})$  and  $C(q)$  are equal to  $D(q)$ ,  $h(q, \dot{q})$  and  $C(q)$  then the above torque equality reduces to

$$D(q) [\ddot{e}(t) + K_v \dot{e}(t) + K_p e(t)] = 0 \quad \text{where } e(t) = q^d(t) - q(t)$$

$D(q)$  is always non-singular.  $K_p$  and  $K_v$  can be chosen appropriately so the characteristic roots of this equation have negative real parts, then  $e(t)$  asymptotically approaches to zero.

Ⓛ

Lagrange-Euler representation becomes inefficient in the computation of joint torques. Paul suggests that one must not design a closed-loop digital controller using L.E representation such as neglecting the velocity related terms of  $h_a(q, \dot{q})$  and the off-diagonal elements of the acceleration related matrix  $D_a(q)$  (Lagrange-Euler).

Then

$$\ddot{z}(t) = \text{diag}[D_a(q)] \left\{ \ddot{q}^d(t) + K_v [\dot{q}^d(t) - \dot{q}(t)] + K_p [q^d(t) - q(t)] \right\} + C_a(q)$$

Computer simulations conducted on this control law shows that terms cannot be neglected when the robot arm is moving at high speeds,



①

Force control, compliness, path trajectory

small;  
iki robotun  
eş zamanlı  
birleşimi tutması

An analogous control law in the joint variable space can be derived from  $N-E$  equations of motion to serve a robot arm, the control law is computed recursively using  $N-E$  equations. The recursive control law can be obtained by substituting  $\dot{q}_i(t)$  into  $N-E$  equations to obtain the necessary joint torque for each actuator.

$$\ddot{q}_i(t) = \ddot{q}_i^d(t) + \sum_{j=1}^n K_v^{ij} [\dot{q}_j^d(t) - \dot{q}_j(t)] + \sum_{j=1}^n K_p^{ij} [q_j^d(t) - q_j(t)]$$

1. The first term generates desired torque for each joint if there is no modelling error and the physical system parameters are known. However there are errors due to backlash gear friction uncertainty about the inertia parameters and time delay into the servo loop so that deviation from the desired joint trajectory will be inevitable.

2. The remaining terms in  $N-E$  eqns of motion will generate the correction torques to compensate for small deviations from the desired ~~trajectory~~ joint trajectory.

## Motion Controls

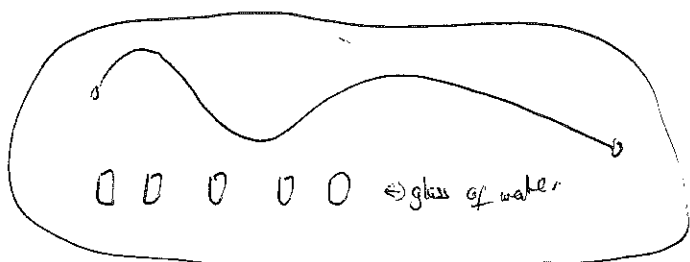
### 1. Joint Motion Control

- 1.1 Joint servo mechanism
- 1.2 computed torque technique
- 1.3 Minimum time control
- 1.4 Variable structure control
- 1.5 Nonlinear decoupled control

### 2. Resolved Motion Controls

- 2.1 Resolved motion rate control
- 2.2 Resolved " acceleration control

Trajectory planning



Trans  $Rot(z, \phi) Rot(y, \theta)$

- ① GROSS MOTION
- ② FINE MOTION

(Need two rotation vectors to carry the glass together with the robot arm)

$${}^0T_6 \quad {}^6T_{Tool} = C_{working}(t) \quad {}^wP_{obj}$$

$\downarrow$        $\downarrow$        $\downarrow$        $\downarrow$   
 $4 \times 4$     $4 \times 4$     $4 \times 4$     $4 \times 4$

Describing

the manipulator hand position + orientation w.r.t the base coordinate frame

the tool position and orientation w.r.t hand coordinate frame  
 It describes the tool end-point whose motion to be controlled.

working coordinate frame of the object w.r.t the base coordinate frame as a function of time

The desired gripping position (or point of manipulation) of the object for the end-effector w.r.t the Working Coordinate frame

$${}^0T_6 = C_{working}(t) \quad {}^wP_{obj} \quad ({}^6T_{Tool})^{-1}$$

$${}^0T_6 ({}^6T_{Tool})_1 = [C_w(t)]_1 ({}^wP_{obj})_1$$

$${}^0T_6 ({}^6T_{Tool})_2 = [C_w(t)]_2 ({}^wP_{obj})_2$$

⋮

$${}^0T_6 ({}^6T_{Tool})_N = [C_w(t)]_N ({}^wP_{obj})_N$$

$${}^{Tool}T_6 T_1 = C_1(t) P_1$$

$${}^{Tool}T_6 T_2 = C_2(t) P_2$$

⋮

$${}^{Tool}T_6 T_N = C_N(t) P_N$$

From the position defined by  $C_i(t) P_i$  we can obtain the distance between consecutive points. If we are given linear  $v$  and angular  $\omega$  velocities we can obtain the time ~~required~~ requested  $t_i$  to move from position  $i$  to  $i+1$

$$T_b^{Tool} T_1 = C_1(t) P_{11}$$

$$T_b^{Tool} T_2 = C_2(t) P_{12}$$

$$P_{12} = C_2^{-1}(t) C_1(t) P_{11} (T_b^{Tool} T_2)^{-1}$$

$$D(\lambda) \quad 0 \leq t = \frac{\lambda}{T} \leq 1$$

Driving function

At position  $i$   $t=0$   $\lambda=0$   $D(0)$   $4 \times 4$

$$P_{i,i+1} = P_i \quad D(1) \neq \quad D(1) = (P_i^{-1} P_{i,i+1})$$

$$P_{i,i} = P_i \quad D(0)$$

$$P_{i,i}^A = A = \begin{bmatrix} n_A & o_A & a_A & p_A \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$= \begin{bmatrix} n_x^A & o_x^A & a_x^A & p_x^A \\ n_y^A & o_y^A & a_y^A & p_y^A \\ n_z^A & o_z^A & a_z^A & p_z^A \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$P_{i,i+1}^B = B = \begin{bmatrix} n_B & o_B & a_B & p_B \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$= \begin{bmatrix} n_x^B & o_x^B & a_x^B & p_x^B \\ n_y^B & o_y^B & a_y^B & p_y^B \\ n_z^B & o_z^B & a_z^B & p_z^B \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$D(1) = \begin{bmatrix} n_A \cdot n_B & n_A \cdot o_B & n_A \cdot a_B & n_A (p_B - p_A) \\ o_A \cdot n_B & o_A \cdot o_B & o_A \cdot a_B & o_A (p_B - p_A) \\ a_A \cdot n_B & a_A \cdot o_B & a_A \cdot a_B & a_A (p_B - p_A) \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

If  $\alpha$  varies linearly with  $t$  then the resultant motion represented by  $D(\alpha)$  will correspond to a constant linear velocity and angular velocities. The translational motion can be represented by

$$L(\alpha) = \begin{bmatrix} 1 & 0 & 0 & \frac{dx}{dt} \\ 0 & 1 & 0 & \frac{dy}{dt} \\ 0 & 0 & 1 & \frac{dz}{dt} \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

and the motion will be along the straight line joining  $P_{i,i}$  to  $P_{i,i+1}$

The first rotational motion can be represented by

$$R_A(\alpha) = \begin{bmatrix} \cos \psi V(\alpha, \theta) (\alpha \theta) \\ -\sin \psi C(\alpha \theta) \\ -C \psi S(\alpha \theta) \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} -\sin \psi V(\alpha \theta) \\ [C \psi V(\alpha \theta) + C(\alpha \theta)] \\ -\sin \psi S(\alpha \theta) \\ 0 \end{bmatrix}$$

$$\begin{bmatrix} C \psi S(\alpha \theta) & 0 \\ \sin \psi S(\alpha \theta) & 0 \\ C(\alpha \theta) & 0 \\ 0 & 1 \end{bmatrix}$$

and it rotates the approach vector at  $P_{i,i}$  to the approach vector at  $P_i$

The second rotational motion represented by  $R_B(\alpha)$  rotates the orientation vector at  $P_{i,i}$  into the orientation vector at  $P_{i,i+1}$  about the tool axis.

$$R_B(\alpha) = \begin{bmatrix} C(\alpha \theta) & -S(\alpha \theta) & 0 & 0 \\ S(\alpha \theta) & C(\alpha \theta) & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$V(\alpha \theta) = V \sin \alpha (\alpha \theta) \\ = 1 - \cos(\alpha \theta)$$

$R_A(\alpha)$  is a rotation of an angle  $\theta$  about the orientation vector of  $P_{i,i}$  which is rotated an angle of  $\psi$  about the approach vector.

$R_B(\alpha)$  is a rotation of an angle  $\phi$  about the approach vector of the foil at  $P_i, i=1$

$$D(\alpha) = \begin{bmatrix} d_n & d_o & d_a & d_p \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

$$d_o = \begin{bmatrix} -s(\alpha\phi) [s^2\psi v(\alpha\theta) + c(\alpha\theta)] + c(\alpha\theta) [-s\psi c\psi v(\alpha\theta)] \\ -s(\alpha\phi) [-s\psi (\psi v(\alpha\theta))] + c(\alpha\theta) [c^2\psi v(\alpha\theta) + c(\alpha\theta)] \\ -s(\alpha\phi) [-c\psi s(\alpha\theta)] + c(\alpha\theta) [-s\psi s(\alpha\theta)] \end{bmatrix}$$

$$d_a = \begin{bmatrix} c\psi s(\alpha\theta) \\ s\psi s(\alpha\theta) \\ c(\alpha\theta) \end{bmatrix}$$

$$d_p = \begin{bmatrix} dx \\ dy \\ dz \end{bmatrix}$$

$$d_n = d_o + d_a$$

$$L(\alpha) = D(\alpha) R_B^{-1}(\alpha) R_A^{-1}(\alpha)$$

$$x = n_A (P_B - P_A)$$

$$y = o_A (P_B - P_A)$$

$$z = a_A (P_B - P_A)$$

$$R_a(\alpha) = L^{-1}(\alpha) D(\alpha) R_B^{-1}(\alpha)$$

$$\psi = \tan^{-1} \left( \frac{o_A \cdot d_B}{n_A \cdot d_B} \right) \quad -\pi \leq \psi \leq \pi$$

$$\theta = \tan^{-1} \left[ \frac{\sqrt{(n_A \cdot a_B)^2 + (o_A \cdot a_B)^2}}{a_A \cdot a_B} \right] \quad 0 \leq \theta \leq \pi$$

$$R_B(\lambda) = P_A^{-1}(\lambda) L^{-1}(\lambda) D(\lambda)$$

$$s\phi = -s\psi c\psi v(\lambda\theta) (\eta_A \cdot \eta_B) + [c^2\psi v(\lambda\theta) + c(\lambda\theta)] (o_A \cdot o_B) - s\psi s(\lambda\theta) (\eta_A \cdot \eta_B)$$

$$c\phi = -s\psi c\psi v(\lambda\theta) (\eta_A \cdot o_B) + [c^2\psi v(\lambda\theta) + c(\lambda\theta)] (o_A \cdot o_B) - s\psi s(\lambda\theta) (\eta_B \cdot o_B)$$

$$\phi = \tan^{-1} \left[ \frac{s\phi}{c\phi} \right] \quad -\pi \leq \phi \leq \pi$$